

Applying Bolt Pre-Loads in LS-DYNA

**The Arup Campus, Blythe Gate, Blythe Valley Park, Solihull, West
Midlands, B90 8AE**

tel: +44 (0) 121 213 3399

email: dyna.support@arup.com

January 2019

There are a number of ways of applying a pre-load to a bolt in LS-DYNA; the following methods are covered in these notes:

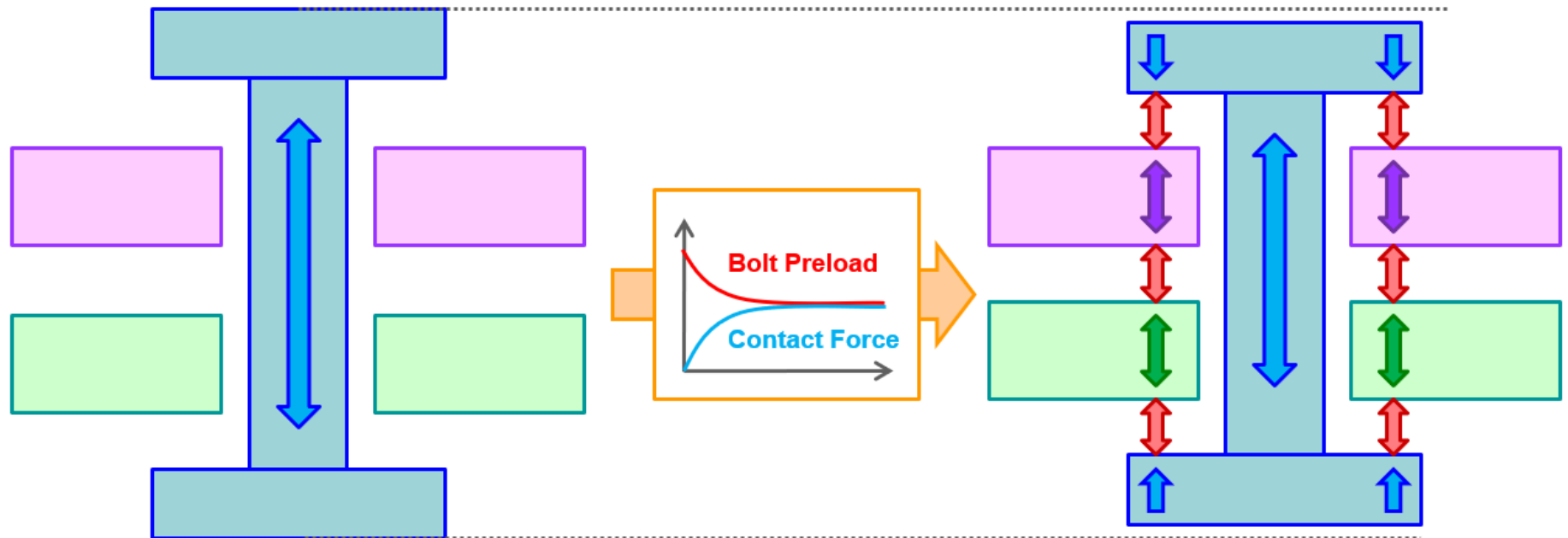
Bolts modelled using solid elements:

- ***INITIAL_STRESS_SECTION** method
- Nut translation method

Bolts modelled using beam elements:

- ***INITIAL_AXIAL_FORCE_BEAM** method (LS971 R4)

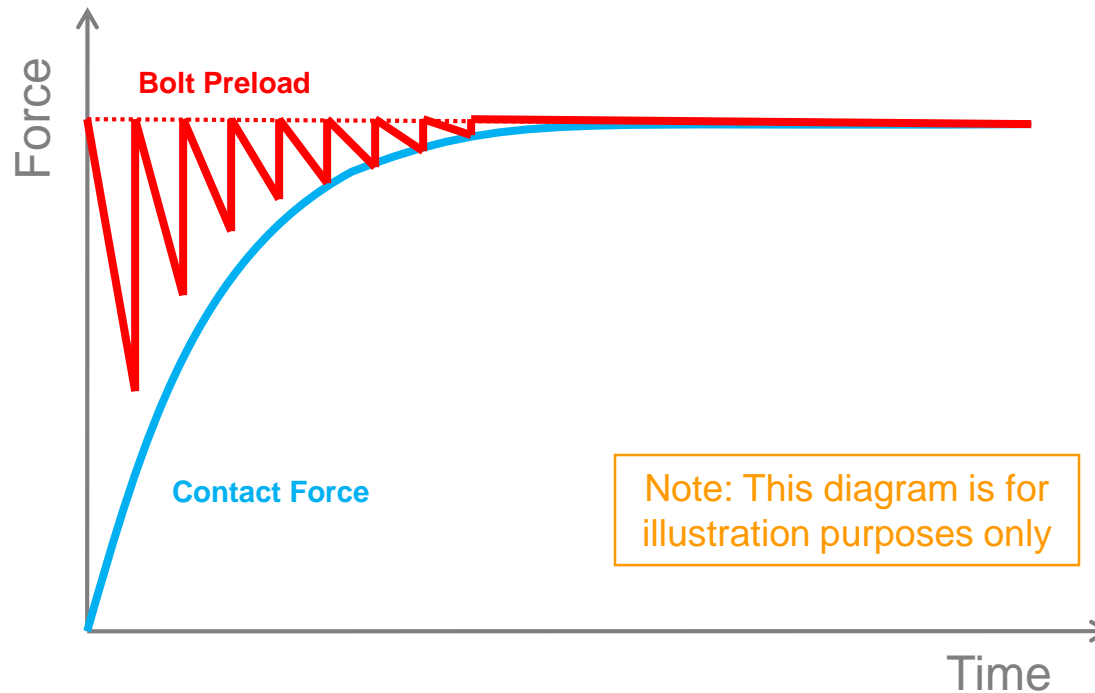
If the standard ***INITIAL_STRESS_SOLID/BEAM** is used to generate the preload in a bolt then some of the load in the bolt will be lost during the start of the analysis as the contact force and stresses in the plates increase to balance out the clamping load in the bolt.



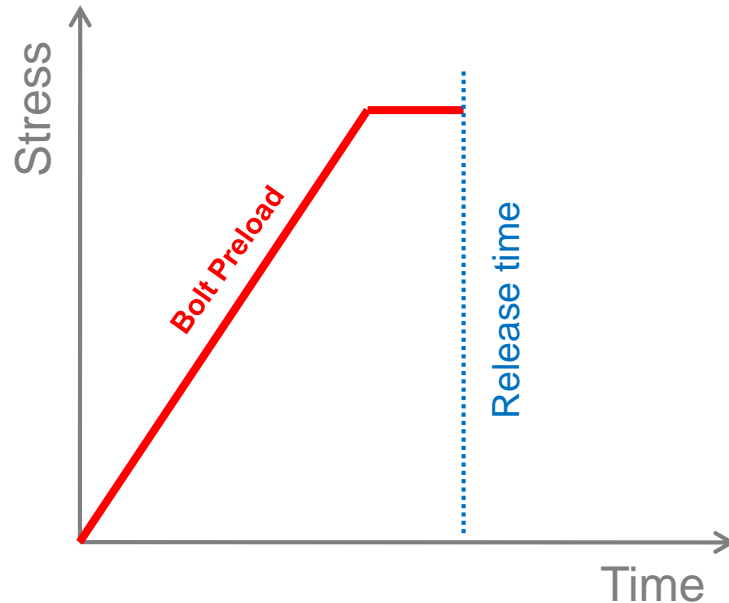
Initial state with the forces
out of balance
(time = 0)

After the pre-load, contact force and
stress in the plates has equalized
(After a few timesteps)

In this method the stress to be applied to the bolt elements is defined using a load curve (stress vs time). During the analysis at the start of each timestep, the stress in the solid elements is reset to this value, and when the load curve ends, the stress in the elements stops being reset. This allows the stress in the bolt to be held at a set value until the pre-load, contact force and stress in the plates have all equalized.



Typically, the stress is ramped up from zero then held constant for a short period time at the start of analysis.



The stress required for a given pre-load can be calculated using;

$$\text{Stress} = \text{Force} / \text{Bolt Area}$$

Note: The bolt cross sectional area is not the area of the real bolt but the area of the elements representing the bolt.

The slower the load can be applied the more accurate the final preload at release will be and the less noise will be generated.

The *INITIAL_STRESS_SECTION card references the:

- *DATABASE_CROSS_SECTION – that cuts through the elements to be loaded
- *DEFINE_CURVE – used to define the pre-load
- *SET_PART – used to define the parts to apply the pre-load to

```
*INITIAL_STRESS_SECTION
```

```
1
```

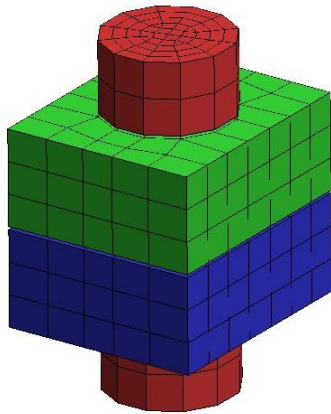
```
1
```

```
1
```

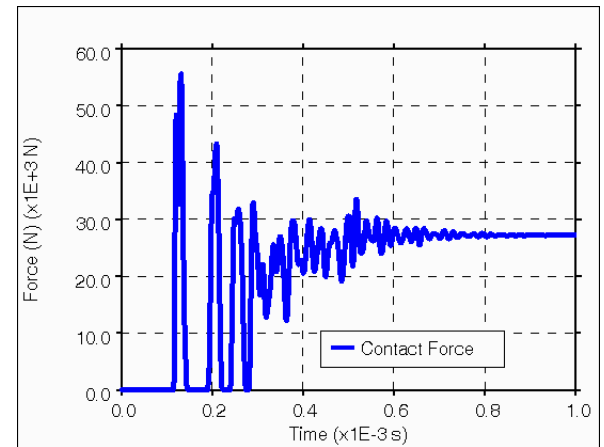
```
1
```

```
0
```

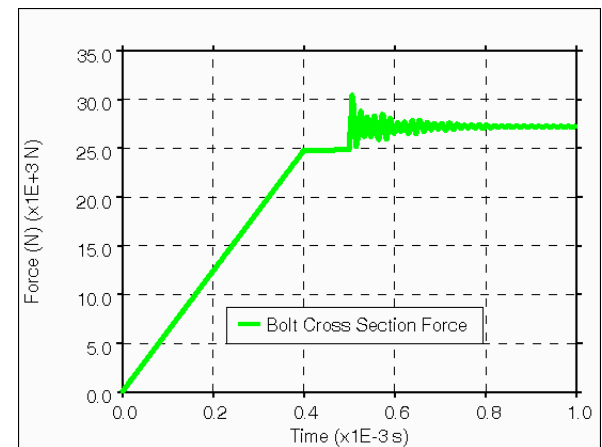
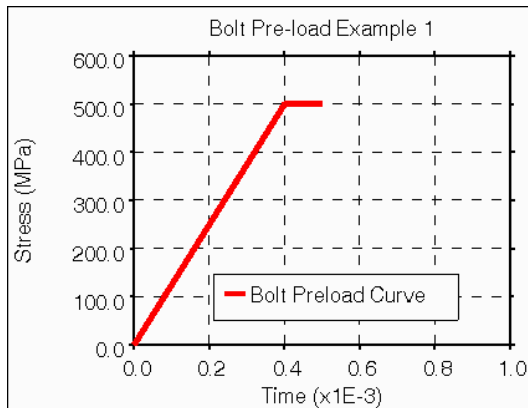
In this example, a pre-loaded bolt is used to clamp two plates together using ***INITIAL_STRESS_SECTION**. The preload ramps up from zero to 500MPa and is released at 0.5msec.



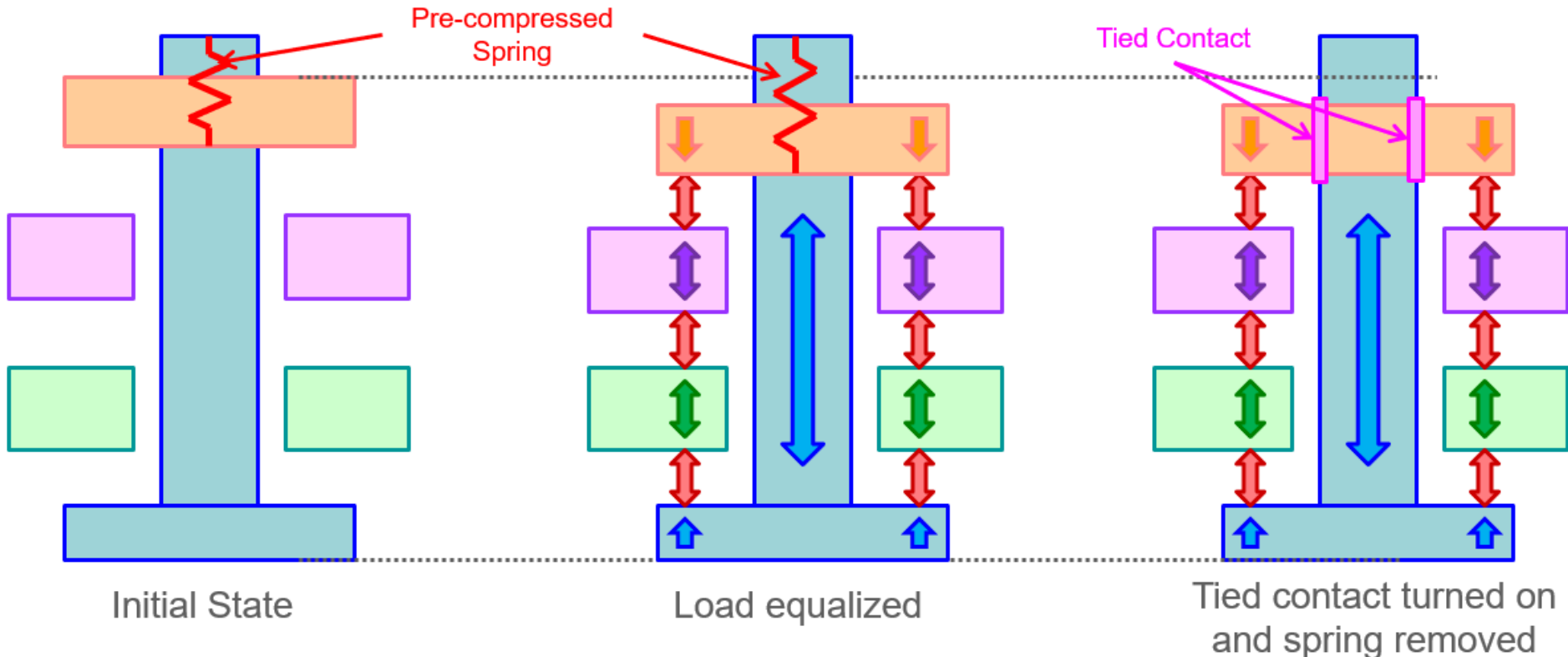
Resulting
Contact Force
between the
plates



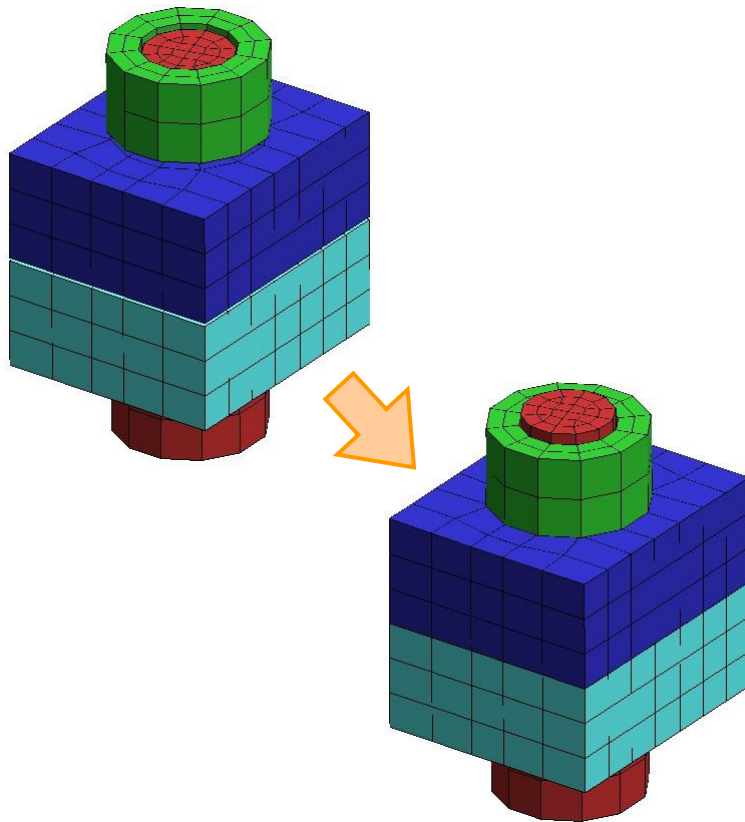
Resulting
Cross-sectional
Force in the bolt



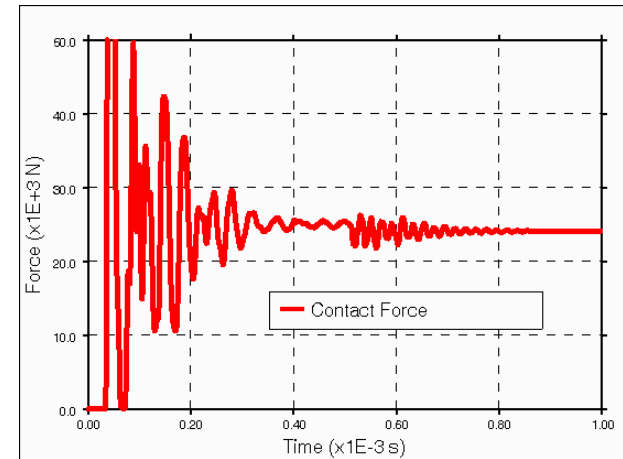
In this method the nut and bolt are modelled separately and at the start of the analysis the plates are clamped together by moving the nut down the shaft of the bolt. The nut is moved using a pre-compressed spring connected between the bolt and the nut. Once the required load is achieved the nut and bolt are fixed together using a tied contact with a particular birth time and then the spring is removed.



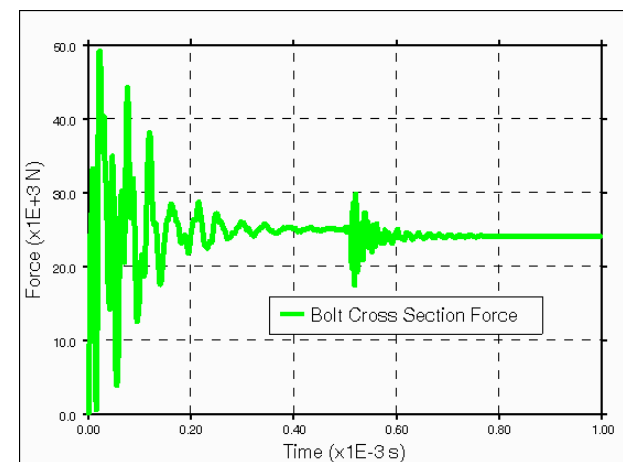
In this example, a pre-loaded bolt is used to clamp two plates together by moving the nut down the bolt shaft. The bolt and nut are locked together using a tied contact at 0.5msec.



Resulting Contact
Force between
the plates



Resulting
Cross-sectional
Force in the bolt



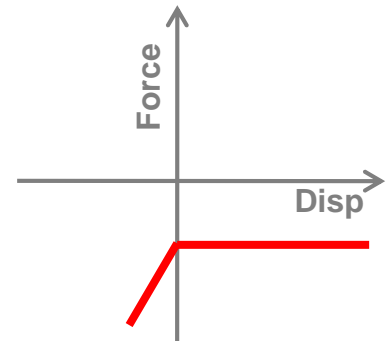
The pre-compressed spring is modelled as a discrete beam (Type 6) using ***MAT_NONLINEAR_ELASTIC_DISCRETE_BEAM** to reference a force vs displacement curve

```
*SECTION_BEAM
    6      6      0.0      0.0      0.0      2.0      0.0
    10.0    0.0      0      0.0      0.0      0.0      0.0
```

```
*MAT_NONLINEAR_ELASTIC_DISCRETE_BEAM_TITLE
Pre-load spring
    4  7.85E-9      1      0      0      0      0      0
    0      0      0      0      0      0
    0.0      0.0      0.0      0.0      0.0      0.0
```

The pre-compression in the spring is modelled by giving it a negative force at zero deflection.

```
*DEFINE_CURVE_TITLE
Spring Load
1      0      0.0      0.0      0.0      0
      -10.000000      -26000.000
           0.0      -25000.000
      100.00000      -25000.000
```



The nut and bolt are locked together using a Tied Contact with a Birth Time (0.5msec).

*CONTACT_TIED_SURFACE_TO_SURFACE_ID_OFFSET

6Nut to Bolt Fixing

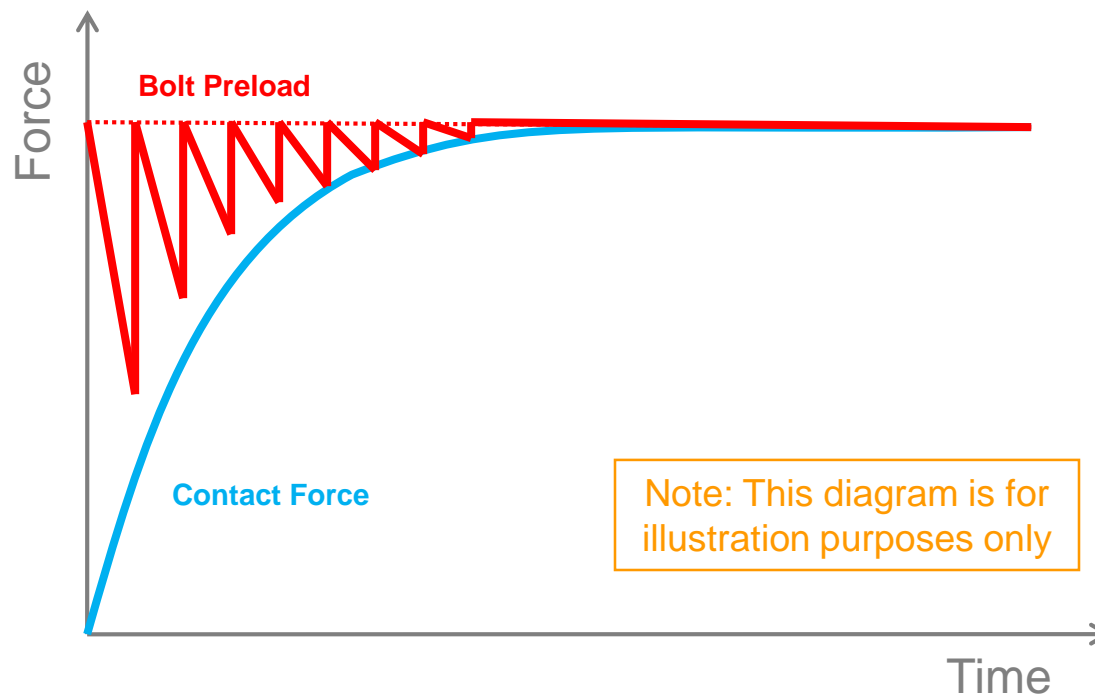
2	1	3	3	0	0	1	1
0.0	0.0	0.0	0.0	0.0	0	5.0E-4	0.0
0.0	0.0	0.5	0.5	0.0	0.0	0.0	0.

The discrete beam (part 6) is removed at time = 0.51msec using the *LOAD_REMOVE_PART card (available for beams in 971 R4.2.1).

*LOAD_REMOVE_PART

6	5.0E-4	5.1E-4	0
---	--------	--------	---

In this method, the pre-load to be applied to the bolt elements is defined using a load curve (axial load vs time). During the analysis, at the start of each timestep, the load in the beam elements is reset to this value, and when the load curve ends, the load in the elements stop being reset. This allows the load in the bolt to be held at a set value until the pre-load, contact force and stress in the plates have all equalized (similar to the *INITIAL_STRESS_SECTION card).

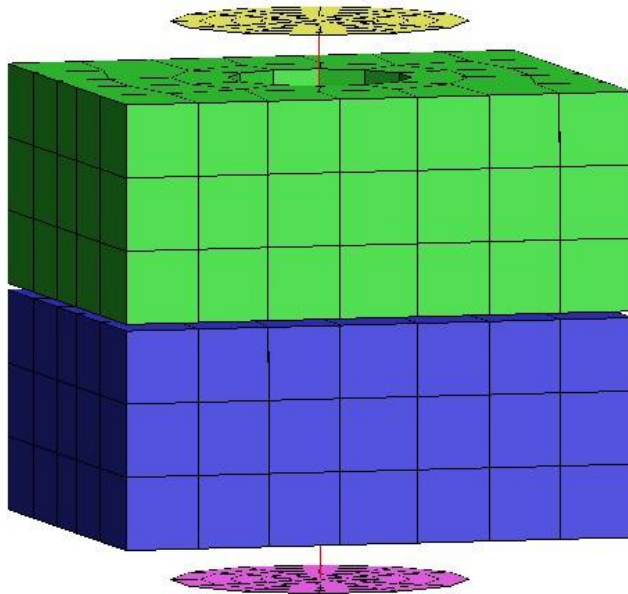


The ***INITIAL_AXIAL_FORCE_BEAM** card works with beam element using element type 9 (Spotweld) and ***MAT_SPOTWELD** (MAT_100). It references the;

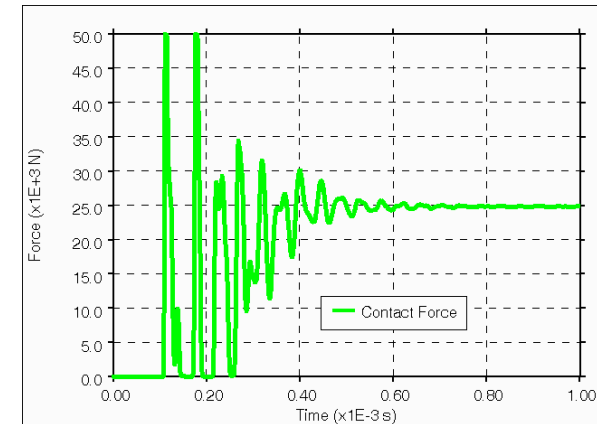
- ***SET_BEAM** – that contains the elements to be loaded
- ***DEFINE_CURVE** – used to define the pre-load

```
*INITIAL_AXIAL_FORCE_BEAM  
    1      1
```

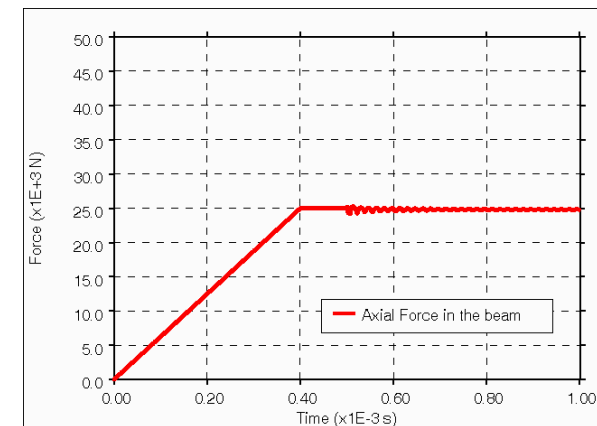
In this example, a pre-loaded bolt is used to clamp two plates together using ***INITIAL_AXIAL_FORCE_BEAM**. The preload ramps up from zero to 25kN and is released at 0.5msec. The heads of the bolt are modelled using rigid elements.



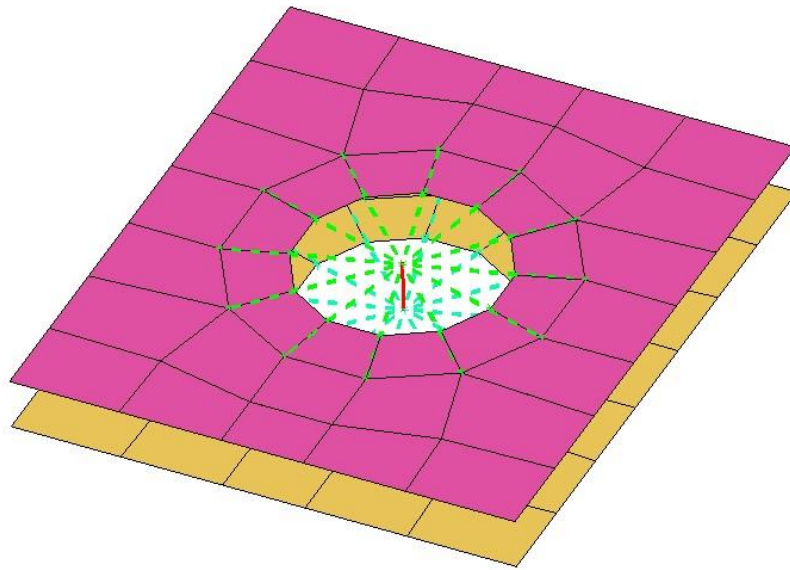
Resulting
Contact Force
between the
plates



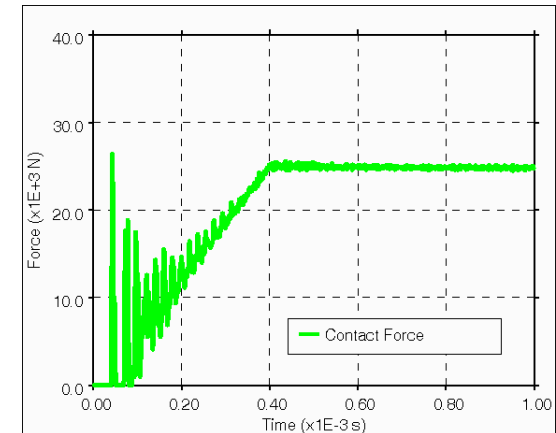
Resulting Axial
Force in the bolt



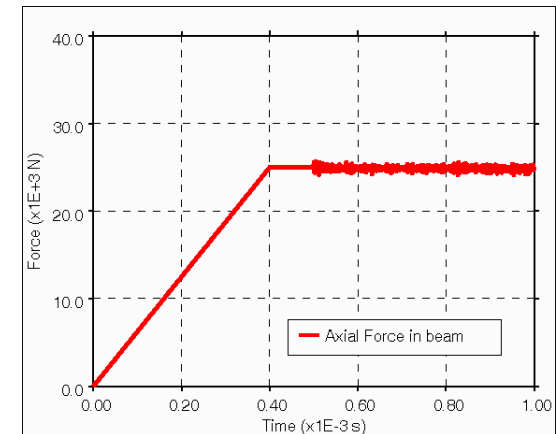
In this example, a pre-loaded bolt is used to clamp two shell element plates together using ***INITIAL_AXIAL_FORCE_BEAM**. The preload ramps up from zero to 25kN and is released at 0.5msec. The ends of the bolt are connected to the plates using ***CONSTRAINED_NODAL_RIGID_BODIES**.



Resulting
Contact Force
between the
plates



Resulting Axial
Force in the bolt



- A small amount of clearance between the parts is recommended, however keeping this to a minimum will reduce the amount of noise.
- A few trial model may need to be run to find the optimum ramp up time for the pre-load (shorter runtime, more noise vs longer runtime, less noise).



www.arup.com/dyna

For more information please contact the following:

UK:

Arup

The Arup Campus
Blythe Valley Park
Solihull, West Midlands
B90 8AE
UK
T +44 (0)121 213 3399
F +44 (0)121 213 3302
dyna.support@arup.com

China:

Arup

39/F-41/F Huai Hai Plaza
Huai Hai Road (M)
Shanghai
China 200031

T +86 21 6126 2875
F +86 21 6126 2882
china.support@arup.com

India:

Arup India Pvt Ltd

Plot No. 39, Ananth Info Park
Hi-Tec City Madhapur Phase 2
Hyderabad - 500081
India

T +91 (0) 40 44369797 / 8

india.support@arup.com